# BACHELOR OF TECHNOLOGY IN MECHANICAL ENGINEERING (COMPUTER INTEGRATED MANUFACTURING) <br> <br> Term-End Examination 

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June, 2010

## BME-020 : KINEMATICS \& DYNAMICS OF MECHANISMS

Time : 3 hours Maximum Marks : 70
Note: Attempt six questions in all. Question number 1 is compulsory and any five questions from the rest. Make suitable assumptions wherever necessary if any.

1. Attempt any five parts of the following: $3 \times 5=15$
(a) Explain higher pairs with the help of suitable sketches and practical examples.
(b) A four bar linkage has lengths of four links as $8,10,12$ and 15 cms . Determine whether it satisfies Grashoff's criteria or not.
(c) A sphere having radius equal to ' $r$ ' is executing pure rolling motion. Find absolute velocities of top point and points on the periphery of horizontal diameter if velocity of centre of the sphere is V .
(d) Explain whirling of shafts.
(e) Explain primary and secondary imbalance of an reciprocating locomotive engine.
(f) Explain where do we use single plane balancing and where do we use two plane balancing ?
2. (a) Classify and write merits and demerits of each cam.
(b) A quick return motion mechanism is shown in Figure 1. The angular velocity of the crank is $20 \mathrm{r} / \mathrm{s}$. Determine angular acceleration of the slotted link.


Figure - 1
3. (a) Explain Chebyshev's spacing of accuracy points.
(b) Design a slider crank mechanism so that the displacement of slider is proportional to the cube of crank rotation in the interval of $30^{\circ} \leq \theta \leq 100^{\circ}$. Assume initial distance of slider equal to 15 cm and final distance to be 10 cm .
4. (a) Find the minimum number of teeth on the pinion to avoid interference when it meshes with a gear with gear ratios $1: 3$ and rack, for the $20^{\circ}$ pressure angle.
(b) In a Hartnell and rotating governor the radius of rotation is 7 cm at 500 rpm . At this speed ball arm is normal and sleeve is in mid position. The sleeve lift is 2 cm with $\pm 5 \%$ change in speed. The mass of the sleeve is 6 kg and the friction is equivalent to 25 N at the sleeve. Ball mass is 2 kg . Determine the following:
(i) spring rate and
(ii) initial compression in spring
5. (a) Explain the effects of imbalance in a rotating system.
(b) A $60^{\circ}$ V-twin engine carries reciprocating mass of 1.5 kg in each cylinder and stroke length is 100 mm . Find maximum and minimum values of primary forces and the corresponding crank positions if engine speed is 1500 rpm .
6. (a) A disc of mass 4 kg is mounted on a shaft of 10 mm diameter at the centre between the two short bearings. The span length is 500 mm . The eccentricity of mass is 2 mm from the geometric centre of the disc. The equivalent viscous damping at the point of disc mounting is $50 \mathrm{~N} \mathrm{sec} / \mathrm{m}$. The angular speed of the shaft is 250 rpm . Determine :
(i) the maximum stress in the shaft, and
(ii) the power required to drive the shaft. Assume E $=2 \times 10^{11} \mathrm{~Pa}$.
7. (a) Explain inversions of double slider crank chain.
$3+3+3+2=11$
(b) Explain vehicle differential with the help of a neat sketch.
(c) Explain force analysis in case of worm gears.
(d) Derive ratio of tensions in a flat belt drive.
8. (a) Find the magnitude and the sense of inertia torque on the crank when it has turned $30^{\circ}$ from inner dead centre for the following data :
$7+4=11$
Reciprocating mass which moves horizontally $\quad=120 \mathrm{~kg}$
Crank length
$=90 \mathrm{~mm}$
Engine speed
$=600 \mathrm{rpm}$
Length of connecting rod
$=450 \mathrm{~mm}$
Mass of connecting rod
$=90 \mathrm{~kg}$
The radius of gyration
$=150 \mathrm{~mm}$
Distance of C. G. from
$=180 \mathrm{~mm}$
small end centre
(b) A mechanical system is shown in Figure 2.

Prove that the natural frequency of the system is :
$\left(\frac{1}{2 \pi} \sqrt{\frac{\mathrm{k}}{\mathrm{m}+\frac{\mathrm{I}}{\mathrm{r}^{2}}}}\right) \mathrm{Hz}$.


Figure-2

